

Basic Utility Vehicle Drivetrain Design Team

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Designer	Design Area
Craig Blough	Electrical Front Wheel Drive
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Challenge

Develop a 3 – wheel vehicle to be driven by polio survivors, landmine survivors, amputees, and others that no longer have use of their legs. In addition to hand controls and affordability, design emphasis is on the steering, front suspension and a third-wheel drive (front wheel). Design vehicle based on re-using the rear axles and suspension of a small pick-up truck. Design for small scale assembly operations in Africa. Minimize factory investment at a volume of one vehicle per day.



What is BUV?

Mission

To improve lives in developing countries by facilitating the spread of simple vehicles that can be assembled “almost anywhere, by almost anyone.”

Vision

The BUV will go:

- ...where the streets have no name
- ...where roads don't exist
- ...where people need hope

Basic Vehicles. Changed Lives.

Goal

To jumpstart an industry to bless the working poor



Acknowledgements

Agricultural & Biological Engineering Support

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- Dr. John Lumkes, Technical Advisor
- Dr. Joe Irudayaraj, Academic Advisor

Organizational Support

- Institute for Affordable Transportation (Will Austin)

Industry Support

- Yanmar
- Mid States Power (Harlan Vanderpool)
- Quality Drive Systems (Roy Navarette)

Others

- ABE Shop: Scott Brand & Gary Williams
- Purdue Quarter-Scale Team
- Purdue Central Machine Shop

Design Objectives

Cost	Performance	Safety	Manufacturing
<ul style="list-style-type: none"> ➢ Minimize total lifetime cost of ownership. ➢ Use off the shelf components ➢ Also use salvaged or pre-existing components when possible ➢ Reduce number of moving parts by avoiding complex gearboxes 	<ul style="list-style-type: none"> ➢ Find the right balance between performance and simplicity ➢ Design drivetrain system with industrial components that can withstand punishment ➢ Allow for easy access of drivetrain components for fine-tuning and maintenance 	<ul style="list-style-type: none"> ➢ Protect driver and passengers from all moving parts and sharp edges ➢ Coordinate placement of components of drivetrain with weight distribution of vehicle ➢ Minimize controls needed to control drivetrain system 	<ul style="list-style-type: none"> ➢ Group drivetrain components into subassemblies that can easily be bolted onto vehicle ➢ Simplify fabrication techniques when possible



Diesel Drivetrain

Design Criteria

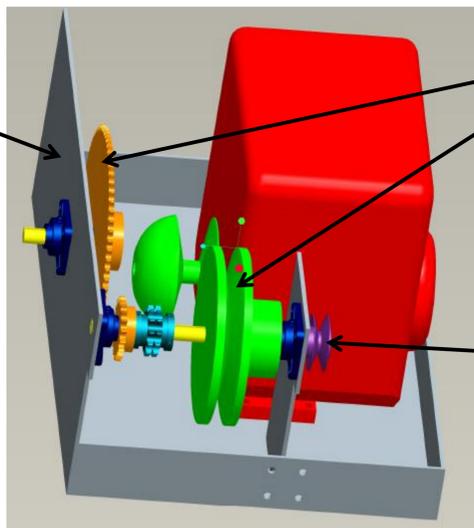
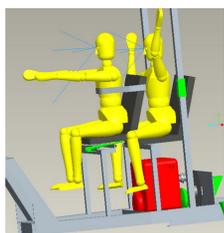
- Must use 10hp Yanmar diesel engine
- Make a modular design that can be integrated with the rest of the vehicle
- Provide a high torque at low speeds and provide 20mph top speed
- Design drivetrain capable of towing 1700lbs including a trailer loaded with 500 lbs
- Must be able to disengage driveline to power PTO pulley
- Simplify drivetrain controls (handicapped driver)



Initial Design

Steel Enclosure

- Keeps drivetrain components safe, dry, and secure while being lightweight
- Modular design that bolts straight to front and rear frame modules
- Provides extra structural rigidity for drivetrain components
- Provides storage for tools, spare parts, and other accessories



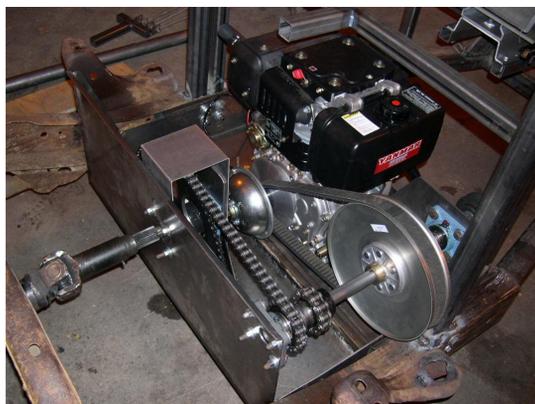
Main Drive System

- A combination of CVT and chain drive system was chosen over a straight belt drive or Tuff Torq transmission.
- This design choice produced a drivetrain package that is simple, lightweight, high performance, and easy to control and maintain
- Engine positioned low and in center of vehicle to improve weight distribution

PTO Drive System

- A roller chain coupler was chosen to disengage power from driveline and power the PTO pulley
- A variable pitch pulley allows for use of a wide range of v belt sizes
- Speed of PTO is adjustable due to CVT
- A friction throttle allows for constant speed while using PTO

Fabrication and Testing



Easy to produce/assemble

- Wire feed and arc welding used to match processes available to theoretical micro-factory
- Used salvaged driveshaft parts from a 1993 S10 to produce custom driveshaft
- Brake and Electrical systems fully integrated into enclosure, allowing for simplified assembly

Optimized weight and performance

- 72T sprocket lightened by 5 lbs through simple machining
- Engine position made adjustable to account for belt wear and wet conditions
- Structural enclosure built with minimum amount of material
- Reinforcements added to increase rigidity after initial testing



Safety

- Aluminum guards used to protect operator and passengers from moving parts and exhaust
- Kill switch lever integrated into operator's station to allow for quick deactivation of engine

BUV Competition Results

- There were issues with the throttle cable, but once fixed, the BUV performed well and obtained 2nd place in the sprint competition
- In the obstacle course, the vehicle did very well until the final obstacle, when the driveshaft fell off of the splined shaft connected to the 72T sprocket

Areas of Improvement

- The driveshaft needs to be lengthened to decrease the yoke angles and subsequent vibrations
- The throttle cable used was too rigid to make extreme bends, a braided cable should be used instead



Calculations

Engine (Yanmar 100V, 10hp diesel engine)			
Rpm	3600	3200	2500
Hp	9.1	8.7	7.4
Torque (ft-lb)	13.3	14.3	15.5

Comet 760 CVT		
High Ratio	1:1	
Low Ratio	5.12:1	
Drive Pulley Diameter	7.25	in
Driven Pulley Diameter	11.26	in
V-Belt Thickness	1.25	in

Chain drive system	
Gear reduction	4:1
Driver Sprocket	18T
Driven Sprocket	72T
Roller Chain Size	#50

Rear Differential	
Gear Reduction	3.73:1

Wheel Dimensions		
Outside Diameter	28	in
Width	6	in

Output at Low Ratio		
Engine Speed	3600	rpm
Total Gear Reduction	76:1	
Max Vehicle Speed	4	mph

Output at High Ratio		
Engine Speed	3600	rpm
Total Gear Reduction	15:1	
Max Vehicle Speed	20	mph

Cost

Engine				
Item/Part	Cost	Qty	Subtotal	
Yanmar 10hp Diesel		350	1	350

CVT - QDS			
Item/Part	Cost	Qty	Subtotal
Comet 760 CVT Drive Pulley	195	1	195
Comet 760 CVT Driven Pulley	185	1	185
Comet CVT Belt	40	1	40

Chain Drive - Surplus Center			
Item/Part	Cost	Qty	Subtotal
1" chain coupling half	16.95	2	33.9
coupling chain	9.75	3	29.25
1" X 3' Keyed Shafting	33.95	1	33.95
Driver 18T unfinished bore	5.25	1	5.25
10' of #50 Chain	18.99	1	18.99
1" 4 bolt flange bearing	8.95	4	35.8

Chain Drive - McMaster - Carr			
Item/Part	Cost	Qty	Subtotal
Driven 72T	87.4	1	87.4
Variable pitch pulley	24.65	1	24.65

CVT			
Item/Part	Cost	Qty	Subtotal
Comet 760 CVT Drive Pulley	195	1	195
Comet 760 CVT Driven Pulley	185	1	185
Comet CVT Belt	40	1	40

TOTAL 1459.19

Electrical Drivetrain

Design Criteria

- To be used in stop and go conditions encountered in trash pickup
- Must be able to propel the vehicle at 1mph in both forward and reverse
- Design as a modular package that can be easily installed on both past and present BUV's
- Simplify drivetrain controls (handicapped driver)



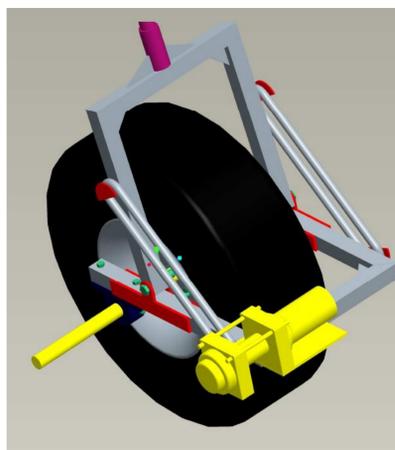
Design Process

Requirements at the Wheel

- Found Requirements at Wheel Hub
- Converted ground speed to rotation at hub
- Found torque required (from previous vehicle testing)
- Combined rpm and torque to find power required

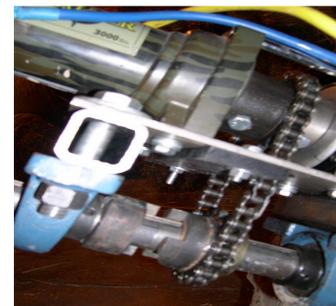
Sizing the Winch

- Sized with power, torque and speed requirements required at the wheel
- Utilized drum diameter to determine maximum possible sprocket size
- Gear reduction chosen to match requirements at the wheel
- When powering the winch, we utilize a starter battery due to the high current required over a short time



Clutching Mechanism

- External mechanism needed due to high freewheel speeds that cause excessive wear to internal parts
- Dog clutch mechanism utilizing lovejoys
- Outer lovejoy keyed to shaft
- Inner lovejoy welded to sprocket, riding bushing
- Outer lovejoy is held normally out with ball bearing, spring and set screw assembly
- Manual engagement by sliding outer lovejoy inward to engage dogs



Part Selection

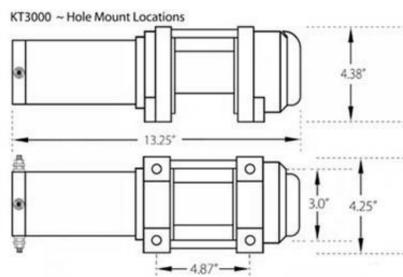
- Utilized common parts for easy sourcing to foreign countries
- 40 roller chain for ease of maintenance and variable sprocket sizes

Calculations

Requirements for Wheel/Winch Interface	
Tire Size	27 in.
Target Speed	1 mph
	12.45 rpm
Drum rotation	5 fpm
	9.55 rpm
Drum Size	2 in.
Gear reduction	1.30 :1

Winch manufacturer specifications

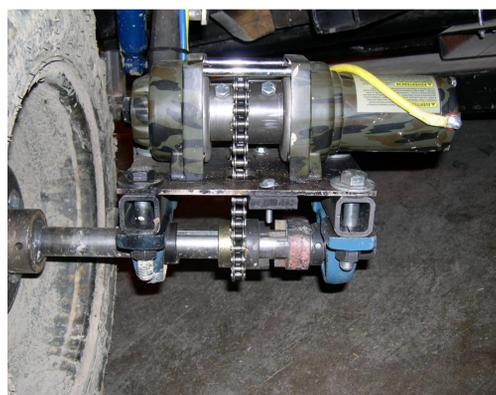
KT3000/KT3000C		Lbs	0	500	1000	2000	3000	
Line speed and Motor current (first layer)	Line speed	Kgs	0	227	454	907	1361	
	Line speed	FPM	27	20	16	9	5	
Motor Current	Line speed	MPM	8.3	6.1	4.9	2.7	1.5	
	Motor Current	Amps	15	40	60	130	190	
Layer of cable	Line pull & cable capacity	Rated line pull per layer	Lbs	3000	2561	2234	1981	1779
			Kgs	1361	1162	1013	899	807
		Cable capacity per layer	Ft.	8.85	10.37	11.89	13.41	14.92
			M	2.7	3.16	3.62	4.1	4.55
Rolling Load Capacities (first layer)	Slope*	10% (4.5°)	Lbs	15075	10251	6428	3854	
		20% (9°)	Kgs	6838	4650	2916	1748	
		40% (18°)						
		100% (45°)						



Fabrication and Testing

Machining

- 5/16" keyway in axle shaft
- Outer lovejoy bored to 1 1/4", keyed
- Inner lovejoy bored to 1 1/2", welded to small sprocket and pressed onto brass bushing
- Sleeve made for axle shaft, drum cut in half, connected to sleeve by shear bolts offset 180°
- Lock collar machined to 1/2" to conserve space inside pillow block
- Chain rub block to prevent inner lovejoy from 'walking out' and engaging shaft

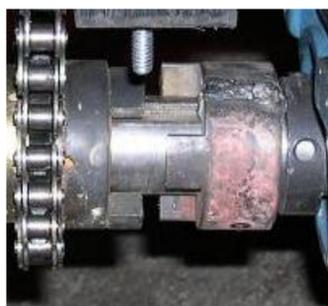


Testing & Competition

- The Electric FWD successfully powered the axle when jacked up and on pavement
- Lovejoy failed on soil with full load
- Failure along plane of set screw and keyway (180° offset)
- New lovejoy was machined out of stronger steel with the key offset 90° from the set screw.
- Second lovejoy failed in competition on soil under full load

Areas of Improvement

- The design was very sound, but failed due to an unexpected part failure
- The Electric FWD successfully powered the axle when jacked up and on pavement,
- Lovejoy failed on soil with full load
- Failure along plane of set screw and keyway (180° offset)
- New lovejoy was machined out of stronger steel with the key offset 90° from the set screw.
- Redesign would include a more heavy duty lovejoy or a sprocket & chain master link clutching assembly.
 - More difficult engagement, but less likelihood of failure
- An FEA analysis of the lovejoy would be effective in analyzing the failure mode.



Final Cost

Item	Qty	Subtotal
ATV Winch	1	\$229.00
Pillow Block	1	\$19.99
Bearing	1	\$11.99
Lovejoy	2	\$12.00
Sprocket	2	\$10.59
40 roller chain	1	\$3.00

TOTAL \$286.57

www.drivebu.org